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Signal and System Theory II

4. Semester, BSc

Solutions

Exercise 1

1	2	3(a)	3(b)	3(c)	4	Exercise
3	4	3	3	6	6	25 Points

1.

$$\text{(1p)} \quad \dot{x} = \bar{v} \tag{1}$$

$$\text{(1p)} \quad \dot{y} = \bar{v}\theta \tag{2}$$

$$\text{(1p)} \quad \dot{\theta} = \frac{\bar{v}}{\ell}u, \quad u = \tan(\phi) \tag{3}$$

Accept the answers if given with v instead of \bar{v} .

2.

$$\begin{bmatrix} x_{k+1} \\ y_{k+1} \\ \theta_{k+1} \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & \delta\bar{v} \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} x_k \\ y_k \\ \theta_k \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \delta\bar{v}/\ell \end{bmatrix} u_k + \begin{bmatrix} \delta\bar{v} \\ 0 \\ 0 \end{bmatrix} \tag{4}$$

The LTI part (A matrix:**2p**, B matrix:**1p**), the constant term at the end (**1p**). OK, to only give matrices and the constant offset of the affine term.

3. (a) The tracking error states are derived as follows.

$$\tilde{x}_{k+1} = x_{k+1} - x_{k+1}^r = x_k + \delta\bar{v} - (x_k^r + \delta\bar{v}) = x_k - x_k^r = \tilde{x}_k \tag{5}$$

$$\tilde{y}_{k+1} = y_{k+1} - y_{k+1}^r = y_k + \delta\bar{v}\theta_k - \bar{y} = \tilde{y}_k + \delta\bar{v}\tilde{\theta}_k \tag{6}$$

$$\tilde{\theta}_{k+1} = \theta_{k+1} - \theta_{k+1}^r = \theta_k + \delta\bar{v}/\ell u_k - 0 = \tilde{\theta}_k + \delta\bar{v}/\ell u_k \tag{7}$$

The resulting error dynamics is then given by

$$\begin{bmatrix} \tilde{x}_{k+1} \\ \tilde{y}_{k+1} \\ \tilde{\theta}_{k+1} \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & \delta\bar{v} \\ 0 & 0 & 1 \end{bmatrix} \begin{bmatrix} \tilde{x}_k \\ \tilde{y}_k \\ \tilde{\theta}_k \end{bmatrix} + \begin{bmatrix} 0 \\ 0 \\ \delta\bar{v}/\ell \end{bmatrix} u_k.$$

The A matrix with the error states (**1p**), B matrix (**1p**). Plugging the values in to show the result (**1p**).

If the values with the equations are directly derived instead of giving the state space, and values are plugged in along the way showing clear progression, (**1p**) per each equation above.

(b) The A matrix has all its eigenvalues at 1 (i.e., $\rho(A) = 1$) (**1p**), therefore the system is not asymptotically stable (stating that it is unstable by inspecting the $(\tilde{y}_k, \tilde{\theta}_k)$ subspace is also correct) (**2p**).

(c)

$$C = [B \ AB \ A^2B](\mathbf{1p}) = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 4 & 8 \\ 2 & 2 & 2 \end{bmatrix} (\mathbf{1p}) \tag{8}$$

The matrix is rank deficient, the system is not controllable (**1p**).

Reachable space spanned by $(0, 1, 0)$ and $(0, 0, 1)$, uncontrollable mode 1 (**1p**).
The system is not stabilisable. (**1p** with justification).

This is caused by the small angle approximation (OK to mention as linearization approximation) and the constant velocity assumption for the input (**1p** for listing at least one reason).

4. Substituting the controller into the system leads to

$$\begin{bmatrix} \tilde{x}_{k+1} \\ \tilde{y}_{k+1} \\ \tilde{\theta}_{k+1} \end{bmatrix} = \begin{bmatrix} 1 & 0 & 0 \\ 0 & 1 & 2 \\ 0 & p & 1 - 1 = 0 \end{bmatrix} \begin{bmatrix} \tilde{x}_k \\ \tilde{y}_k \\ \tilde{\theta}_k \end{bmatrix} \quad (9)$$

The controllable subspace spans the $\tilde{y}, \tilde{\theta}$ states, thus we ignore the pole at 1, which is uncontrollable. The controllable subspace is then

$$\begin{bmatrix} \tilde{y}_{k+1} \\ \tilde{\theta}_{k+1} \end{bmatrix} = \begin{bmatrix} 1 & 2 \\ p & 0 \end{bmatrix} \begin{bmatrix} \tilde{y}_k \\ \tilde{\theta}_k \end{bmatrix}. \quad (10)$$

There should be an explanation of what the closed-loop dynamics is and how we can get the desired poles, e.g., writing the A matrix etc., similar to the description above. Explanation until this point is (**3p**) (**1p** for describing the closed loop dynamics, **2p** for describing how to get the poles using the gain p). We want both poles of the characteristic polynomial at 0.5.

$$(\mathbf{2p}) \quad \det(sI - A) = s(s - 1) - 2p = (s - 0.5)^2 \implies p = -0.125$$

Solving the characteristic polynomial for the 3x3 A matrix is also fine.

The error states \tilde{y} and $\tilde{\theta}$ in this discrete-time model are asymptotically stable. The uncontrollable state \tilde{x}_k , on the other hand is stable, but not asymptotically stable. (**1p**)

Exercise 2

1(a)	1(b)	1(c)	1(d)	2(a)	2(b)	2(c)	Exercise
3	3	4	3	4	4	4	25 Points

1. (a) The observability matrix is:

$$O = \begin{bmatrix} C \\ CA \end{bmatrix} = \begin{bmatrix} 1 & 0 \\ a & b \end{bmatrix} \text{ (1 p)}$$

For O to have full rank, we need $b \neq 0$, all other parameters a, c, d are irrelevant for observability and therefore can be chosen arbitrarily **(2 p)**.

- (b) The controllability matrix is:

$$P = [B \quad AB] = \begin{bmatrix} 1 & a \\ 0 & c \end{bmatrix} \text{ (1 p)}$$

For P to have full rank, we need $c \neq 0$, all other parameters a, b, d are irrelevant for controllability and therefore can be chosen arbitrarily **(2 p)**.

- (c) The closed loop system is:

$$\hat{A} = A + BK = \begin{bmatrix} 1 & 0 \\ -2 & -1 \end{bmatrix} + [k \quad 0] \begin{bmatrix} 1 \\ 0 \end{bmatrix} = \begin{bmatrix} k+1 & 0 \\ -2 & -1 \end{bmatrix} \text{ (1 p)}$$

For the closed loop system to be asymptotically stable we need that all eigenvalues are strictly smaller than 0 **(1 p)**. As \hat{A} is a triangular matrix, the eigenvalues are the elements along the diagonal. Therefore, we require the following condition: $k+1 < 0$, i.e. $k < -1$ **(1 p)**.

As for the fastest convergence rate, since the eigenvalue -1 is fixed, it will determine the fastest possible rate of convergence. This is achieved when the second eigenvalue is at least as fast, that is when $k+1 \leq -1$, i.e. $k \leq -2$ **(1 p)**.

- (d) From the system dynamics we directly obtain that $Kx(t) = ky(t)$ **(1 p)**, so the controller can be directly implemented using a output feedback **(2 p)**.

Note that the fact that for $b = 0$ the system is unobservable and therefore the observer will not converge (Part (a)) is irrelevant in this case.

2. (a) From the characteristic polynomial $\det(\lambda I - A)$, one can easily derive that the eigenvalues of the system are -4, 2 and -8 **(2 p)**. As one of the eigenvalues (i.e. $\lambda = 2$) is strictly positive **(1 p)**, we can conclude that the system is not stable **(1 p)**.

- (b) To determine whether the system is detectable, we have to perform the detectability test on the unstable eigenvalues and verify whether we achieve full rank. The detectability test is:

$$\text{rank} \left(\begin{bmatrix} \lambda I - A \\ C \end{bmatrix} \right)$$

For $\lambda = 2$:

$$\begin{bmatrix} 2I - A \\ C \end{bmatrix} = \begin{bmatrix} 0 & -6 & 0 \\ 0 & 10 & 0 \\ -6 & 4 & 6 \\ 2 & 0 & 0 \end{bmatrix} \text{ (1 p)}$$

This matrix has clearly full rank (rank = 3) **(1 p)**.

As the detectability test has full rank with $\lambda = 2$, the system is detectable **(2 p)**.

Note that however the system is not observable as its observability matrix O defined as:

$$O = \begin{bmatrix} C \\ CA \\ CA^2 \end{bmatrix} = \begin{bmatrix} 2 & 0 & 0 \\ 4 & 12 & 0 \\ 8 & -72 & 0 \end{bmatrix}$$

has clearly $\text{rank}(O) = 2$ (not full rank).

- (c) To determine whether the system is stabilisable, we have to perform the stabilisability test on the unstable eigenvalues and verify whether we achieve full rank. The stabilisability test is:

$$\text{rank}([\lambda I - A \quad B])$$

For $\lambda = 2$:

$$[2I - A \quad B] = \begin{bmatrix} 0 & -6 & 0 & 4 \\ 0 & 10 & 0 & 0 \\ -6 & 4 & 6 & 0 \end{bmatrix} \text{ (1 p)}$$

This matrix has clearly full rank (rank = 3) **(1 p)**.

As the stabilisability test has full rank with $\lambda = 2$, the system is stabilisable **(2 p)**.

Note that however the system is not controllable as its controllability matrix P defined as:

$$P = [B \quad AB \quad A^2B] = \begin{bmatrix} 4 & 8 & 16 \\ 0 & 0 & 0 \\ 0 & 24 & 144 \end{bmatrix}$$

has clearly $\text{rank}(P) = 2$ (not full rank).

Exercise 3

1(a)	1(b)	1(c)	1(d)	2(a)	2(b)	Exercise
4	4	4	8	3	2	25 Points

1. (a) From the differential equation, we identify system states as $x_1(t) = z(t)$ and $x_2(t) = \dot{z}(t)$. Then we have

$$\begin{aligned} \dot{x}_1(t) &= \dot{z}(t) = x_2(t), \\ \dot{x}_2(t) &= \ddot{z}(t) = -a\dot{z}(t) - bz(t) + u(t) = -ax_2(t) - bx_1(t) + u(t) \end{aligned}$$

Using that $\dot{z}(t)$ is the measured system output, finally we can write the state-space form of the system as

$$\begin{aligned} \dot{x}(t) &= \underbrace{\begin{bmatrix} 0 & 1 \\ -b & -a \end{bmatrix}}_A x(t) + \underbrace{\begin{bmatrix} 0 \\ 1 \end{bmatrix}}_B u(t), \\ y(t) &= \underbrace{\begin{bmatrix} 0 & 1 \end{bmatrix}}_C x(t) + \underbrace{0}_D u(t) \end{aligned}$$

(give **(1 p.)** for each matrix correctly written; **(4 p.)** in total)

- (b) Since $z(0) = \dot{z}(0) = 0$, $x(0) = 0$ as well, i.e., the system has a zero initial condition (**(1 p.)**).

i. Solution 1:

From the state space form we can obtain the transfer function from input $u(t)$ to output $y(t)$ by using $G(s) = C(sI - A)^{-1}B$ (**(1 p.)**). Hence,

$$G(s) = \underbrace{\begin{bmatrix} 0 & 1 \end{bmatrix} \begin{bmatrix} s & -1 \\ b & s+a \end{bmatrix}^{-1} \begin{bmatrix} 0 \\ 1 \end{bmatrix}}_{(1 \text{ p.})} = \underbrace{\frac{s}{s^2 + as + b}}_{(1 \text{ p.})}$$

ii. Solution 2

The transfer function from input $u(t)$ to output $y(t)$ can be obtained by using $y(t) = \dot{z}(t)$, the differential equation describing the system and solving the system of equations in the Laplace domain (**(1 p.)**). Hence,

$$\begin{aligned} s^2z(s) - s\dot{z}(0) - z(0) + a(sz(s) - z(0)) + bz(s) &= u(s), \\ y(s) &= sz(s) - z(0). \end{aligned}$$

(**(1 p.)** for the correct Laplace domain expressions.) Then $G(s) = \frac{y(s)}{u(s)} = \frac{sz(s)}{u(s)} = \frac{s}{s^2+as+b}$ (**(1 p.)** for the correct expression).

- (c) For $(a, b) = (4, 0)$ we have $G(s) = \frac{1}{s+4}$. We can use a final value theorem to obtain the impulse response $\lim_{t \rightarrow \infty} y(t) = \lim_{s \rightarrow 0} sG(s) = 0$. (**(1 p.)** for Laplace transform of an impulse being 1, and **(1 p.)** for the correct steady-state response)

For $(a, b) = (0, 4)$ we have $G(s) = \frac{s}{s^2+4}$. The final value theorem cannot be used in this case. However, the inverse Laplace transform can be used. Using

that the Laplace transform of the step is $\frac{1}{s}$ we have $\mathcal{L}^{-1}\{\frac{1}{s^2+4}\} = \frac{1}{2} \sin(2t)$. ((**1 p.**) for Laplace transform of an step, and (**1 p.**) for the correct steady-state response)

- (d) For $a = 2$, $b = -3$, we have $\frac{s}{s^2+2s-3} = \frac{s}{(s+3)(s-1)}$. Hence the system has a stable ($s = -3$) and unstable ($s = 1$) pole and a zero at $s = 0$. Hence, for $\omega \in [0, 1)$ rad/s the magnitude characteristic rises with 20dB/dec, for $\omega \in [1, 3)$ rad/s it is constant, and finally, for $\omega \in [3, \infty)$ rad/s it drops with 20 dB/dec. The phase characteristic rises at $\omega = 1$ rad/s because of the unstable pole and then drops because of the stable pole at $\omega = 3$ rad/s (**1 p.**). Hence, the transfer function corresponds to the bode plot G_2 . (**1 p.**)

For $a = 4$, $b = 3$, we have $\frac{s}{s^2+4s+3} = \frac{s}{(s+3)(s+1)}$. Hence the system has two stable poles ($s = -3$ and $s = -1$) and a zero at $s = 0$. The magnitude characteristic is the same as for the previous case. However, the phase characteristic drops twice for $\frac{\pi}{2}$ (**1 p.**). Hence the correct pairing is with the transfer function G_1 (**1 p.**).

For $a = 3$, $b = 0$ we have $\frac{s}{s^2+3s} = \frac{1}{(s+3)}$. Hence the magnitude characteristic is flat until $\omega = 3$ rad/s when it drops with 20 dB/dec and where the phase characteristic drops for $\frac{\pi}{2}$ (**1 p.**). Hence, the correct pairing is with the transfer function G_4 (**1 p.**).

For $a = 0.1$ and $b = 9$ we have $\frac{s}{s^2+0.1s+9}$. Hence, the system has a pair of stable conjugate-complex poles ($\omega_{1,2} = \frac{-0.1 \pm i\sqrt{35.99}}{2}$) that exhibit resonance in the magnitude characteristic and drop the phase characteristic for π at $\omega \approx 3$ rad/s (**1 p.**). Hence, the correct pairing is with the transfer function G_3 (**1 p.**).

2. (a) The zero of the system is -1 and the poles are $\frac{-3 \pm 3\sqrt{3}}{2}$. Hence, the system has neither zeros nor poles with non-negative real part. Consequently $P = 0$ (**1 p.**). Since $P = 0$, according to the Nyquist criterion, for stability we need $N = Z = 0$ encirclements of the point $-1/K$ (**1 p.**). When $K = -2$, then $-\frac{1}{K} = 0.5$. Since there are no encirclements of that point, the system is stable (**1 p.**).
- (b) If the K keeps decreasing, it will enter the area in which the point $-\frac{1}{K}$ is encircled twice ($N = 2$, thus $Z = 2$) (**1 p.**), hence the system becomes unstable (**1 p.**).

Exercise 4

1	2	3	4(a)	4(b)	4(c)	4(d)	Exercise
3	4	4	4	2	4	4	25 Points

- Nonlinear: Due to x_1x_2 terms [1pt]
 - Time-invariant as dynamics do not explicitly depend on time [1pt]
 - Autonomous as there is no input and it is time invariant [1pt]

Grading: **1 pt** for each question if answer correct and at least one correct justification.

- For equilibria we require the state space equations $f(x) = \begin{bmatrix} -x_1 + bx_1x_2 \\ -2x_2 - bx_1x_2 + a \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$. [1pt]

- One equilibrium at $\bar{x} = \begin{bmatrix} 0 \\ \frac{a}{2} \end{bmatrix}$ [1pt]
- A second equilibrium at: $\tilde{x} = \begin{bmatrix} a - \frac{2}{b} \\ \frac{1}{b} \end{bmatrix}$ [2pts]

- The Jacobian of the system is [1pt] :

$$A = \frac{\partial f}{\partial x}(x) = \begin{bmatrix} -1 + bx_2 & bx_1 \\ -bx_2 & -bx_1 - 2 \end{bmatrix}$$

For $\bar{x} = \begin{bmatrix} 0 \\ \frac{a}{2} \end{bmatrix}$ the Jacobian is:

$$A = \begin{bmatrix} -1 + \frac{a}{2}b & 0 \\ -\frac{a}{2}b & -2 \end{bmatrix}$$

The eigenvalues are $\lambda_1 = -2$ and $\lambda_2 = \frac{1}{2}ab - 1$. Asymptotically stable if $ab < 2$ [1pt] and unstable if $ab > 2$. Note that the case $ab = 2$ is excluded by the assumption $a \neq \frac{2}{b}$.

Equilibrium at $\tilde{x} = \begin{bmatrix} a - \frac{2}{b} \\ \frac{1}{b} \end{bmatrix}$:

$$A = \begin{bmatrix} 0 & ab - 2 \\ -1 & -ab \end{bmatrix}$$

The characteristic polynomial is: $\lambda^2 + \lambda ab + ab - 2 = 0$ [1pt]. The equilibrium is asymptotically stable if and only if all coefficients have the same sign, hence asymptotically stable if $ab > 2$ [1pt] and unstable if $ab < 2$.

4. The equilibrium is $\begin{bmatrix} 0 \\ \frac{a}{2} \end{bmatrix}$ and the Lyapunov function is:

$$V(x) = x_2 + x_1 - \frac{a}{2} \ln(x_2)$$

- (a) We check that the set $S = \{x \mid x_1 \geq 0, x_2 \geq 0\}$ is invariant by analyzing the vector fields at the set boundary:
- For $x_1 = 0$: $\dot{x}_1 = 0, \dot{x}_2 = -2x_2 + a$ [1pt], so x_1 cannot become negative [1pt].
 - If $x_2 = 0$: $\dot{x}_1 = -x_1, \dot{x}_2 = a > 0$ [1pt], thus the system always remains in the positive quadrant and S is an invariant set [1pt].
- (b) From the Jacobian we get directly, $\lambda_1 = -2$ and $\lambda_2 = 0$ [1pt]. The stability analysis is inconclusive due to the zero eigenvalue [1pt].
- (c) Analyze $\frac{d}{dt}V(x(t))$:

$$\begin{aligned} \frac{d}{dt}V(x(t)) &= \begin{bmatrix} \frac{\partial V}{\partial x_1} & \frac{\partial V}{\partial x_2} \end{bmatrix} \begin{bmatrix} f_1 \\ f_2 \end{bmatrix} \\ &= \begin{bmatrix} 1 & 1 - \frac{a}{2x_2} \end{bmatrix} \begin{bmatrix} -x_1 + bx_1x_2 \\ -2x_2 - bx_1x_2 + a \end{bmatrix} \quad [1pt] \\ &= -\frac{(2x_2 - a)^2}{2x_2} \quad [1pt] \text{for analysing signs.} \end{aligned}$$

Thus, $\forall x(t) \in S_c$ we have that $\frac{d}{dt}V(x(t)) \leq 0$ [1pt]. Consequently, $V(x(t))$ will remain smaller than c and the set S_c is invariant [1pt].

- (d) LaSalle's theorem states that all trajectories initialized in S_c converge to the largest invariant set in $M = \{x \in S_c \mid \nabla V(x)f(x) = 0\} = \{x \in S_c \mid x_2 = \frac{a}{2}\}$ [2pts]. To stay on M , x_2 must remain equal to $\frac{a}{2}$ which is only the case if $\dot{x}_2 = 0$. And since on M we have that $\dot{x}_2 = x_1$, it follows that we must have $x_1 = 0$, hence, we must be at the equilibrium \bar{x} . As \bar{x} is the only invariant set in S_c , all trajectories starting in S_c will converge to \bar{x} [2pts].